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TM-1794

## TESLA Test Cell Cryostat Support Post Thermal and Structural Analysis

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### INTRODUCTION

TeV Superconducting Linear Accelerator (TESLA) cryostats consist of eight, 1-meter-long radio frequency (RF) cavity modules cryogenically connected in series with one focusing quadrupole. Each module contains one, 9-cell superconducting RF cavity operating at 1.3 GHz in a 1.8K helium bath. Individual modules are self-contained in the sense that they have their own input couplers, high order mode couplers, and tuning mechanisms. Services common to the entire cryostat consist of 70K and 4.5K thermal radiation shields, shield supply and return lines, a 1.8K helium supply line, and a gas helium return pipe. All cavity modules, the quadrupole, and cryogenic services are contained in a single 12-meter-long vacuum vessel.

The goal of the present work on TESLA is the successful fabrication and test of four complete cryostat assemblies. These cryostats will be installed in a string, cooled to operating temperature, and powered. This test will address problems which may arise when modules are installed in a tunnel environment. It will also permit testing of the basic cooling concepts, measurement of static heat losses, and measurement of the RF performance of all cavities.<sup>1</sup>

All of the current design options utilize a post-type suspension system modeled after that developed for SSC collider dipoles. However, rather than a reentrant design like those in early SSC prototypes<sup>2-5</sup>, this support uses a single

filament wound composite tube. This latter design has recently been adopted for production SSC collider dipoles.<sup>6</sup>

Any successful design must be structurally adequate to meet the static and dynamic loads which occur during fabrication, shipping, installation, and operation. It must have low thermal conductivity to insulate the 1.8K helium volume from heat conducted from 300K and must be manufacturable at low cost. This report attempts to summarize the thermal and structural analysis leading to the selection of a candidate design for supports suitable for use in TESLA test cell cryostats.

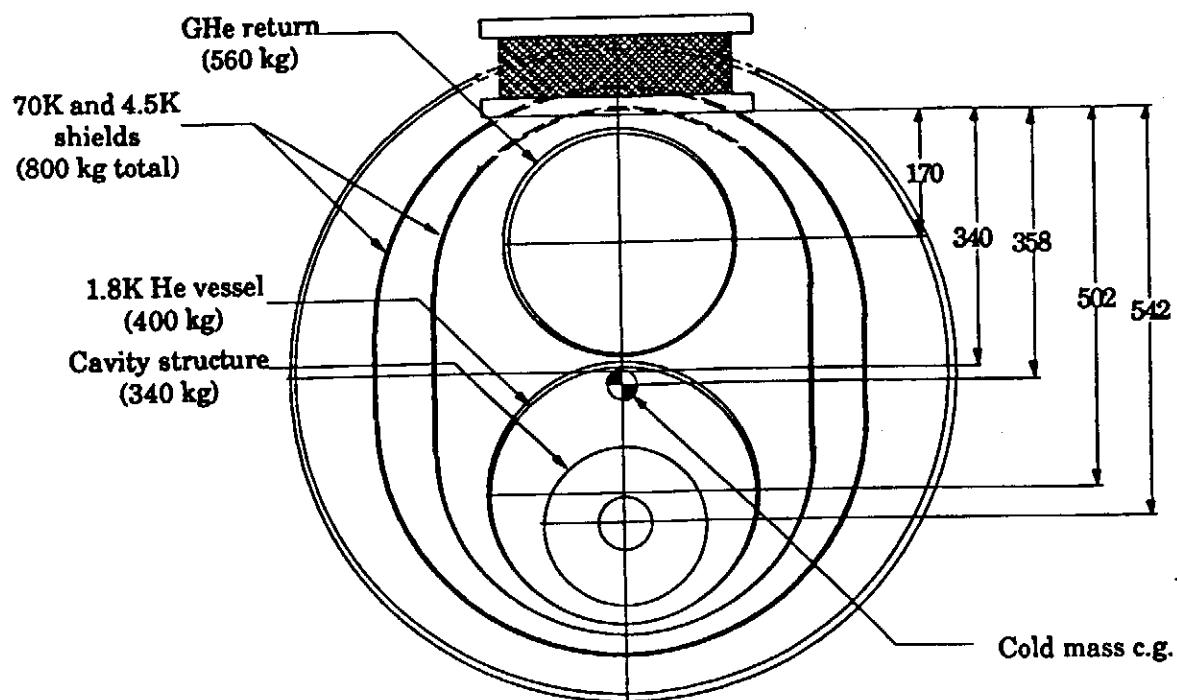
## DESIGN OVERVIEW

There are two conceptual designs being discussed with respect to support post mounting. One uses supports located on top of the vacuum vessel such that the cold mass hangs from the support. The second uses supports located on the bottom of the vacuum vessel such that the cold mass rests on the support. This second concept is the more conventional of the two options, however, in principal there is no reason that hanging the cold mass from the support poses any inherent installation or reliability problems. The advantage to the hanging concept is that it provides a readily accessible place from which to gather direct alignment data when the complete cryostat is installed in the test string. There are two substantive disadvantages. First, a cryostat with top-mounted supports requires reinforcing rings around the vacuum vessel at each support location to support the weight of the hanging assembly. Second, it moves the cavity centerline further from the fixed support base making it more sensitive to displacements occurring due to cooldown and to the action of external forces, e.g. forces acting through the input coupler. These effects will be discussed later in this report. Figures 1 and 2 illustrate the differences between these two design options. The cross section shown is that currently being developed at DESY and INFN.

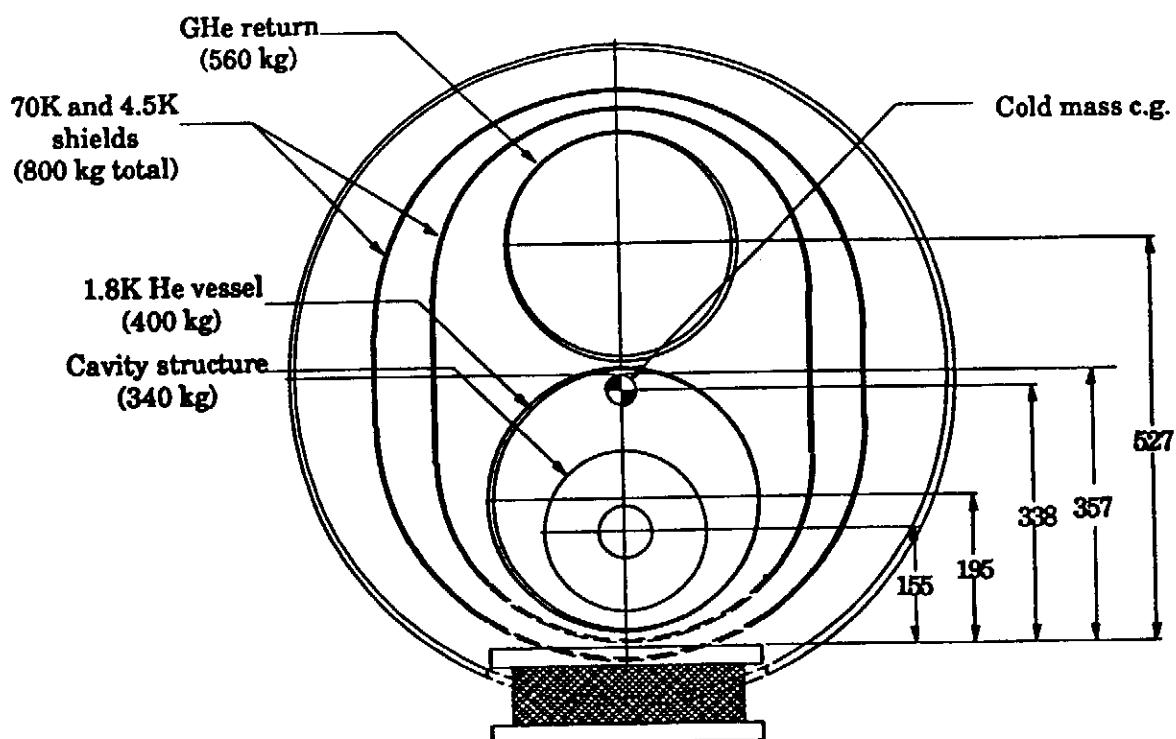
There are also two conceptual designs being discussed with respect to the number of supports. One uses three supports as a means by which to minimize the cost of support assemblies and the cost of the vacuum vessel and thermal radiation shields. The other uses four supports to minimize axial contraction during cooldown. These latter two conceptual design differences have little effect on the analysis presented here, but will be discussed in more detail later in this report.

## DESIGN ANALYSIS

There is little debate about the conceptual design of the support post itself. All of the design options being discussed utilize a single tube support developed as an alternative to the reentrant supports used in SSC collider dipole magnets.<sup>2-6</sup> The single tube support was developed primarily to reduce magnet cost.



**Figure 1. DESY Cross Section - Top Mounted Support**



**Figure 2. DESY Cross Section - Bottom Mounted Support**

## Thermal Path Length Optimization

The design of a TESLA cryostat support begins with a thermal analysis to determine the relative position of the thermal intercepts. There are intercepts at 300K, 70K, 4.5K, and 1.8K. The 300K and 1.8K positions are fixed at the ends of each support. The 70K and 4.5K intercept locations may be chosen anywhere along the length of the support. Their position is dictated by constraints on the allowable static heat load. For this analysis it is assumed that the goal is to minimize the refrigeration power required at room temperature. The heat load at each thermal intercept is translated into a corresponding refrigeration requirement at room temperature by using an expression for the ideal work defined by Carnot and a realistic refrigerator efficiency. The Carnot efficiency is given by the following.

$$\text{Carnot efficiency} = T / (300 - T) \quad [1]$$

where T is expressed in K.

Realistic refrigerator efficiencies are more difficult to estimate. Experiences at Fermilab, with the Tevatron refrigeration system, and at DESY, with the HERA system, indicate that reasonable refrigerator efficiencies are 20% at 70K and 4.5K and 10% at 1.8K. Combining these with the Carnot efficiencies results in the following room temperature loads. The results are expressed in watts per watt (W/W), e.g. 328 watts of power at room temperature are required to produce one watt of refrigeration at 4.5K.

**Table 1. Room Temperature Refrigeration Requirements**

T	Carnot eff	Refrig eff	Combined eff	r.t. W/W
70K	30.43%	20%	6.09%	16
4.5K	1.52%	20%	0.30%	328
1.8K	0.60%	10%	0.06%	1657

Figure 3 is a thermal model of a single tube support illustrating the pertinent analysis parameters. The optimal thermal path lengths (l) as fractions of the total support height are functions of the thermal intercept temperatures, material thermal conductivity, and tube cross sectional area (A). Ideally, there are thermal resistances at each intercept and at the cold mass connection. For the sake of this and subsequent analyses, these are assumed to be perfect connections. In reality, this assumption leads to a conservative result, i.e. actual heat loads, particularly to 1.8K are somewhat smaller than calculated values. The material assumed for the support is S-glass in an epoxy matrix. The thermal conductivity curve for this material is shown in figure 4. The dimension

nomenclature and the results from this analysis are shown in figures 5 and 6 respectively.

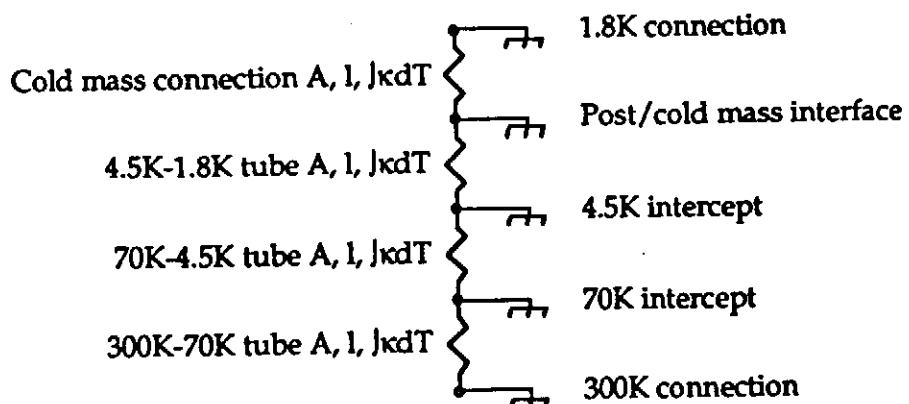


Figure 3. Single Tube Support Thermal Model

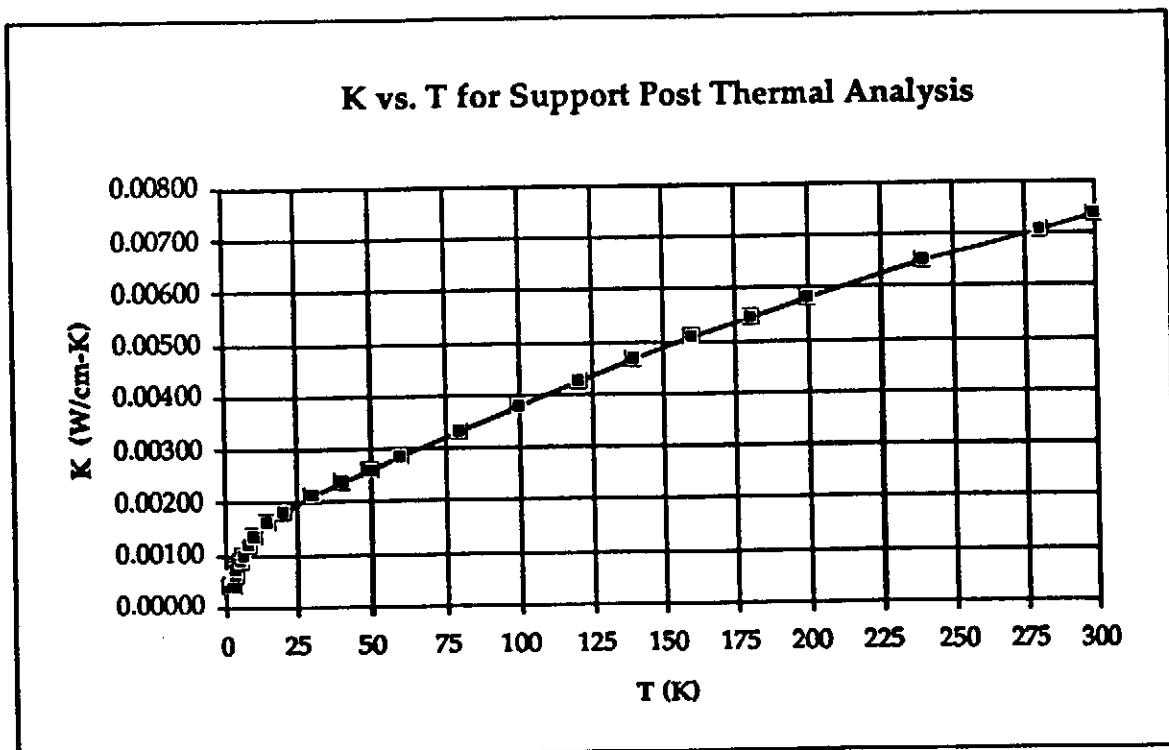
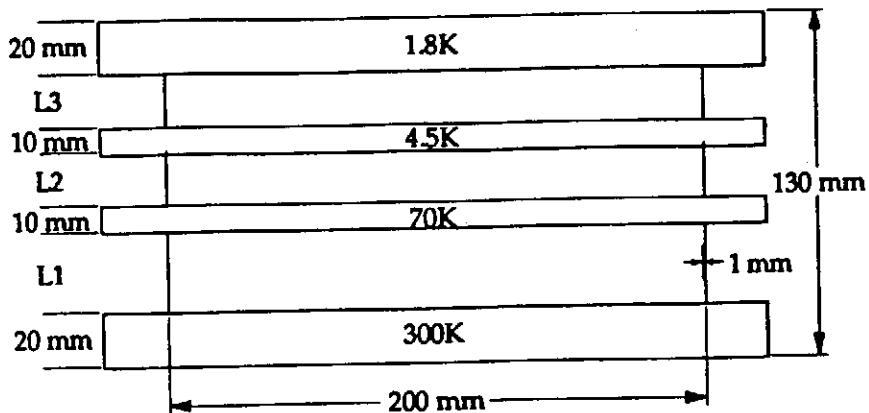
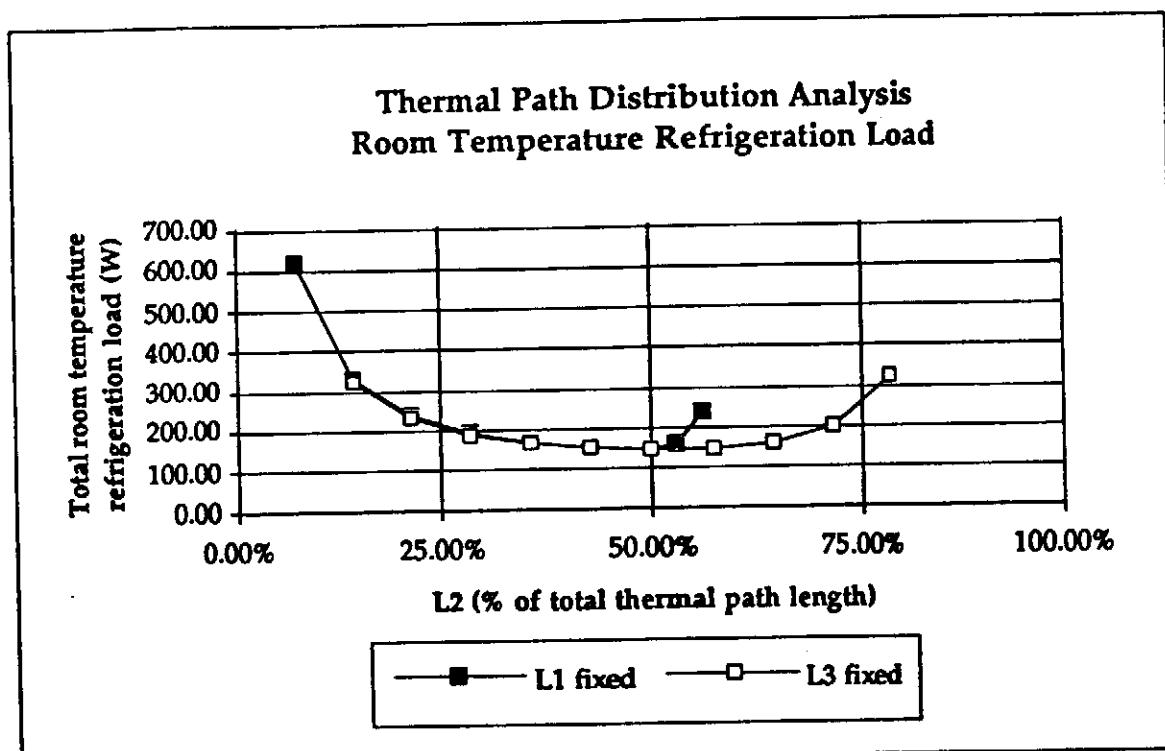


Figure 4. Support Post Material Thermal Conductivity vs. Temperature



**Figure 5. Thermal Path Length Analysis Support Dimension Nomenclature**



**Figure 6. Thermal Path Length Analysis Results**

Using figure 6, the minimum room temperature heat load occurs when L2, the 70K to 4.5K thermal path length, is 50% of the total thermal path length. This is the case when either L1 or L3 is held fixed. It is clear from figure 6 that the total room temperature heat load is rather insensitive to changes in L2 at its optimum, permitting relative freedom in positioning thermal intercepts as long as L2 is approximately 50% of the total thermal path length. This is especially true if L3 is fixed and only L1 and L2 are adjusted.

The path length optimization analysis also indicates that L3 should be as small as practical, approximately 5 mm for the support shown in figure 5. This is somewhat small when fabrication of the supports and subsequent assembly into the cryostat are considered. For mechanical attachment considerations, subsequent analyses will use 10 mm for L3, the thermal path length between 4.5K and 1.8K.

### Detailed Thermal and Structural Analysis

The thermal path length optimization defines the position of thermal intercepts along the length of the support without regard for specified heat loads or structural requirements. The analysis of an actual support structure must consider the thermal and structural load constraints simultaneously. These are generally at odds with one another, that is low heat load implies low strength while high structural strength implies increased heat load. The budgeted heat loads and structural constraints are given in tables 2 and 3 respectively.<sup>1</sup>

**Table 2. Budgeted Heat Loads per Meter of Cryostat Length (W/m)**

70K	4.5K	1.8K
0.5	0.2	0.05

**Table 3. Structural Load Constraints**

Load Direction	Load
Vertical	1.0 g
Lateral	1.0 g
Axial	1.8 g

Given the room temperature heat load conversions in table 1 and the budgeted heat loads in table 2, an equivalent room temperature heat load budget may be defined by the following.

$$Q_{r.t} = (0.5 \times 16) + (0.2 \times 328) + (0.05 \times 1657) = 156.45 \text{ W/m} \quad [2]$$

For a 12 meter long cryostat, this results in the following allowable heat loads per support.

**Table 4. Budgeted Heat Loads per Support (W)**

# of supports	70K	4.5K	1.8K	r.t. equivalent
3	2.00	0.80	0.20	626
4	1.50	0.60	0.15	469

The weights of all cold mass components are given in figures 1 and 2. These weights are estimated and include the GHe return pipe (560 kg), 70K and 4.5K shields (800 kg), 1.8K He vessels (400 kg), and RF cavity structures (340 kg). Weights for small components are included in the totals for each sub-system. There is an additional, estimated 140 kg quadrupole at one end of the cryostat assemble. The entire suspended cold mass weight is therefore 2240 kg. It is sufficiently accurate for the sake of this analysis to assume the weight is uniformly distributed among all supports. For the three and four support options, this results in the following lateral load per support.

**Table 5. Lateral Loads per Support (kg)**

# of supports	Lateral load (kg)
3	746.7
4	560.0

Using figures 1 and 2, the center of gravity of the cold mass assembly is nearly equidistant from the 1.8K surface of the post in both cases, 358 mm in the case of the top mounted support and 338 mm in the bottom mounted case. At this point in the design, there is enough uncertainty in the weights and final geometry to allow us to treat them as equal.

A physical envelope to limit the scope of the optimization study was chosen for the support post structure. Figure 7 illustrates the nomenclature used in the support post structural and thermal analysis. Table 6 lists all of the parameters and constraints used to define the dimensions for any particular analysis iteration. Note that the 70K to 4.5K thermal path length ( $L_{Thrm2}$ ) is half the total thermal path length to minimize the room temperature heat load per the path length analysis above. The analysis program is capable of optimizing the wall thickness of each tube section, i.e. T1, T2, and T3 in figure 7. Ideally the tube can be machined with different thicknesses to take advantage of the decreasing bending stress along the support length. However, this has little effect on heat loads and decreases the lateral natural frequency of the support. Uniform wall thicknesses are assumed here to minimize machining cost on the complete assembly.

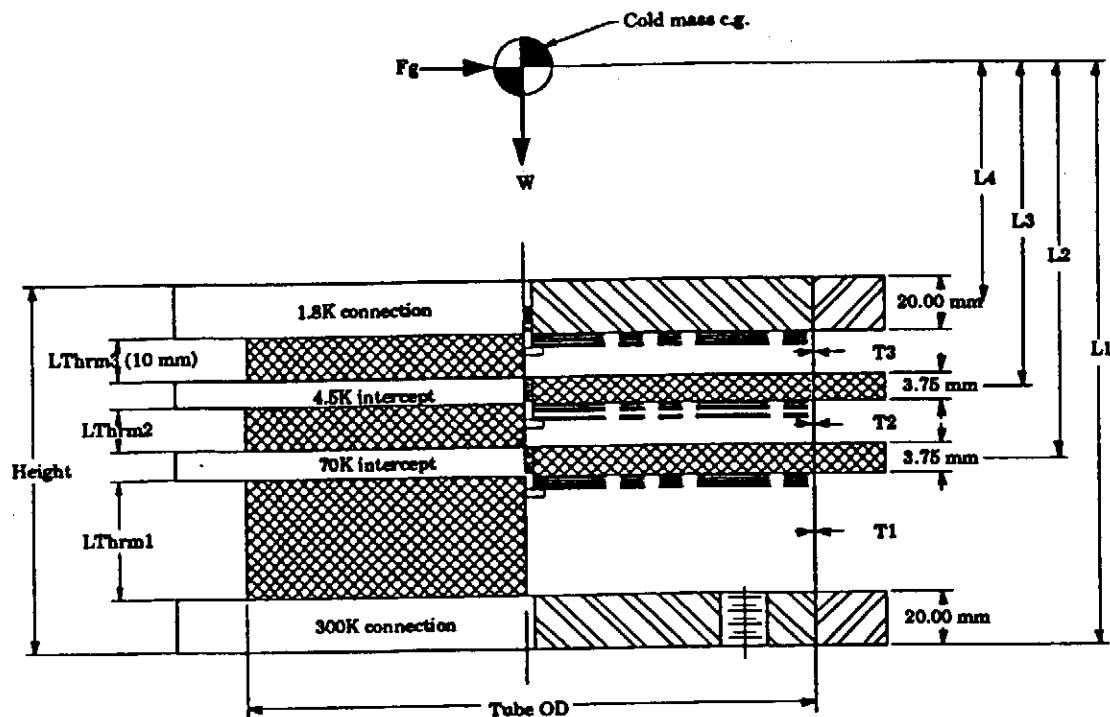


Figure 7. Support Analysis Nomenclature

Table 6. Support Post Dimensional Parameters and Constraints

Overall height range	100 mm - 250 mm
Outside diameter range	150 mm - 300 mm
Wall thickness	Uniform to satisfy strength, safety factor = 2
300K and 1.8K flange thkns	20 mm
70K and 4.5K intercept thkns	10 mm
LThrm1 (300K-70K path len)	Height - 60 mm - LThrm2 - LThrm3
LThrm2 (70K-4.5K path len)	(Height - 60 mm) / 2
LThrm3 (4.5K-1.8K path len)	10 mm
L1	Height + 348 mm
L2	L1 - LThrm1 - 30 mm
L3	L2 - LThrm2 - 20 mm
L4	358 mm
Fg (lateral c.g. force)	746.7 kg (3 supports), 560.0 kg (4 supports)
W	-Fg

Table 7 lists the applicable thermal and structural properties for the composite material assumed for these analyses.

**Table 7. Support Post Material Thermal and Structural Properties**

$\lambda_{kdT}$ (300K-70K)	1.249 W/cm
$\lambda_{kdT}$ (70K-4.5K)	0.146 W/cm
$\lambda_{kdT}$ (4.5K-1.8K)	0.0015 W/cm
E (Young's modulus)	27.58 GPa
n (Poisson's ratio)	0.2
G (Shear modulus)	2.62 GPa
$s_{ult}$ (ultimate tensile and compressive strength)	275.8 MPa
$t_{ult}$ (ultimate shear strength)	137.9 MPa

**Thermal and Structural Analysis Results**

Analyses were performed over the height and diameter ranges listed in table 6. The complete results for these analyses are shown in appendix A, table A-1 and figures A-1 through A-5 for the case of a cryostat with 3 supports and appendix B, table B-1 and figures B-1 through B-5 for the case using 4 supports. The analysis yields tube stresses, heat loads to 70K, 4.5K, and 1.8K, the equivalent room temperature refrigeration power required to meet these heat loads, the cold mass lateral deflection when subject to the lateral load ( $F_g$ ), and an estimate of the lateral resonant frequency. Although not explicitly specified in the design requirements it is thought that the support resonant frequency should be above 10 Hz and below 25 Hz to minimize susceptibility to ground motion and electrically induced vibrations (50 Hz power) respectively. For this analysis a 12.5 to 18.75 Hz band has been defined within which the calculated resonance should fall. These values are 25% above and below the 10 Hz and 25 Hz limits respectively.

From tables A-1 and B-1 a total of five supports satisfy all of the above criteria, i.e. 70K, 4.5K, and 1.8K heat loads, structural constraints, and the constraint on resonant frequency. These are summarized in table 8 and shown in tables A-1 and B-1 in shaded, bold type.

Several things become clear when looking at the results shown in these tables and figures. The supports which satisfy all of the criteria outlined here tend to be larger than those in previous conceptual designs. Typical diameters have varied between 200 and 300 mm. Heights have varied between 100 and 190 mm. The larger diameters here are required to meet the proposed resonant frequency constraint. The greater heights are required to meet the heat load budget, given the increased diameters. Tube stresses are typically well below the allowable (defined as the ultimate strength derated by a safety factor of two). This is due to the fact that the governing structural criteria is elastic stability, i.e.

the tubes are sized to prevent local buckling of the material which occurs at stresses well below that causing tensile or compressive failure.<sup>3,4</sup>

**Table 8. Analysis Summary of Candidate Supports**  
 (see appendices A and B for details)  
 (heat loads are per support)

# of supports	Height (mm)	Diameter (mm)	Q70 (W)	Q4.5 (W)	Q1.8 (W)	Q r.t. (W)	Nat freq (Hz)
3	220	275	1.96	0.20	0.02	130	12.5
3	230	300	1.93	0.20	0.02	130	12.9
3	240	300	1.82	0.19	0.02	125	12.6
4	240	275	1.50	0.16	0.02	103	12.6
4	250	300	1.49	0.16	0.02	105	13.1

These results also indicate that the specified heat loads to 70K, 4.5K, and 1.8K are not consistent with minimizing the room temperature heat load. For example, for the first entry in table 8, the calculated heat loads which meet the 70K, 4.5K, and 1.8K specifications and minimize room temperature heat load are 1.96 W to 70K, 0.20 W to 4.5K, and 0.02 W to 1.8K resulting in a room temperature load of 130 W, nearly a factor of five below the value listed in table 4. If minimized room temperature heat load is, in fact, a viable specification, it would help the design process to broaden the range of individual thermal station heat loads. One could then, for example, look for a design solution that results in a resonant frequency more toward the middle of the 12.5 to 18.75 Hz band, or define a stiffness specification in some other way. All of the cases presented in appendices A and B have room temperature heat loads below the budget shown in table 4, some with 70K heat loads five times the 70K budget.

As an example, suppose that, rather than the absolute limits on the 70K, 4.5K, and 1.8K heat loads in table 2, the specification were rewritten as shown in table 9.

**Table 9. Alternate Specification for Budgeted Heat Loads per Meter of Cryostat Length (W/m)**

70K	4.5K	1.8K
< 1.0	< 0.25	< 0.025

This results in a new set of specifications for the heat load per support shown in table 10.

**Table 10. Budgeted Heat Loads per Support (W)  
Using Alternate Specification in Table 9**

# of supports	70K	4.5K	1.8K	r.t. equivalent
3	< 4.00	< 1.00	< 0.10	558 max
4	< 3.00	< 0.75	< 0.075	418 max

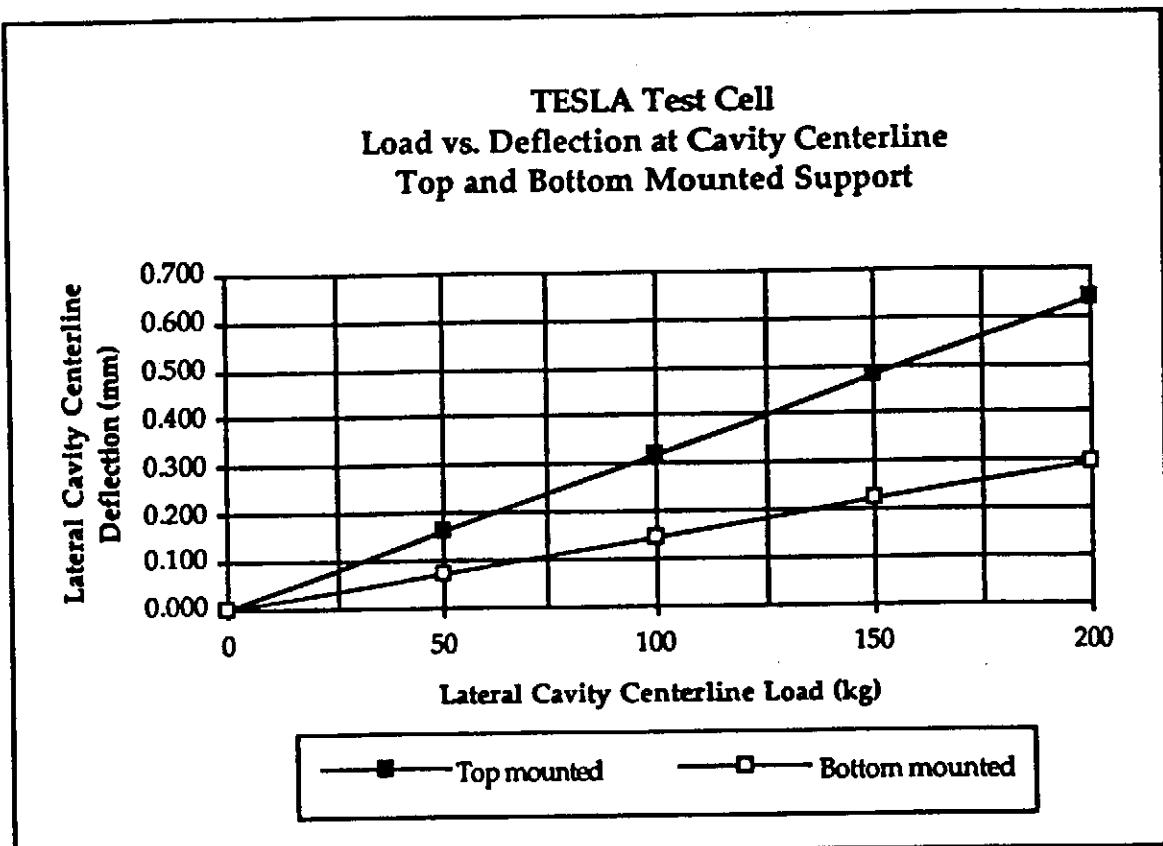
Even with all the above heat loads at their upper limit, the room temperature load is less than that given by equation [2] and listed in table 4. What this alternate specification does, however, is open up much of tables A-1 and B-1 for the selection of suitable support alternatives.

### Top vs. Bottom Mounted Supports

As stated in the introduction to this report, there are no inherent difficulties with either top or bottom mounted supports from the standpoint of assembly or long term stability. Each has its merits and its drawbacks. There is however, one substantial difference, and that is in the deflection of the cavity centerline when subjected to external forces. Forces will act on the beam tube during cooldown by virtue of support from the gas helium tube and from thermal contraction which occurs in the input coupler. Although the magnitude is unknown, some relative differences can be calculated for each option. Figure 8 illustrates the beam tube centerline deflection for the support case highlighted in table A-1, page 3, and the cryostat dimensions in figures 1 and 2. Due to the distance the cavity lies from the support post base, the beam tube deflection in the top mounted case is more than a factor of two greater than in the bottom mounted case. Given the tight alignment tolerances required in the final installation, it seems that the design should strive for any alternatives which increase stiffness and thereby minimize deflections under the influence of outside forces.

### SUMMARY

The analysis presented here is meant as a guide to the design of TESLA test cell cryostat supports. As in any complex device, there are many factors to be considered. Hopefully, most have been covered here, but some have been mentioned only briefly, e.g. alignment, cost, reliability, and ease of manufacture. Using criteria discussed throughout this report, a few conclusions can be drawn.



**Figure 8. Load vs. Deflection for Top and Bottom Mounted Supports**

First, assuming that minimizing the room temperature heat load is a viable basis for design development, the original heat load budgets to 70K, 4.5K, and 1.8K might warrant revision. Second, a firm specification on the suspension system lateral stiffness would be a useful means by which to assure a good overall design, not just one which meets the heat load budget. Third, thermally and structurally, there is no significant advantage to a cryostat using three or four supports. This is not surprising due to the fact that the structural and thermal analyses are largely linear. Cost and thermal contraction issues will likely play more significant roles in this choice. Finally, although potentially more practical in the alignment process, top mounted supports are probably not the best choice if one hopes to minimize lateral deflections of the cavity beam line during cooldown, alignment, and other operations which may subject the cavity or helium vessel to external forces.

The goal of this report has been to focus on issues critical to the development of a suspension system which addresses all of the pertinent design issues. Hopefully, it can serve as a guide for continued suspension system development and be useful as a tool to select or discount various conceptual design options.

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6. Nicol, T.H., "SSC 50mm Collider Dipole Cryostat Single Tube Support Post Conceptual Design and Analysis", Fermilab TM-1745, SSCL-N-765, July 9, 1991, presented at the 1992 Industrial International Symposium on the Super Collider (IISSC), March 4-6, 1992, New Orleans, LA.

## **APPENDIX A**

### **Thermal and Structural Analysis Results 3 Supports per Cryostat**

TESLA Test Cell Support Analysis  
3 supports per axial  
T-Nicoll, Fermilab, August 1992

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Refrigeration System			
Wet to Wet	Off	Cry.	Wet
T [K]		[W]	[W]
70	20%	16	70
4.5	20%	328	4.5
1.8	10%	1657	1.8

Detailed Building Support									
T	C	Q1							
[K]	[W]								
70	200	33	70	200	33	4.5	80	263	1.8
4.5	80	263	4.5	80	263	1.8	20	331	1.8

Detailed Building Support									
T	C	Q1							
[K]	[W]								
70	200	33	70	200	33	4.5	80	263	1.8
4.5	80	263	4.5	80	263	1.8	20	331	1.8

Detailed Building Support									
T	C	Q1	Q1	Q1	Q1	Q1	Q1	Q1	Q1
[K]	[W]	[W]	[W]	[W]	[W]	[W]	[W]	[W]	[W]
70	200	33	70	200	33	4.5	80	263	1.8
4.5	80	263	4.5	80	263	1.8	20	331	1.8
1.8	10%	1657	1.8	20	331	1.8	20	331	1.8
100.0	175.0	448.0	413.0	383.0	358.0	10.0	20.0	1.575	97.4
110.0	175.0	458.0	418.0	383.0	358.0	15.0	25.0	1.600	98.4
120.0	175.0	468.0	423.0	383.0	358.0	20.0	30.0	1.600	99.4
130.0	175.0	478.0	428.0	383.0	358.0	25.0	35.0	1.626	100.4
140.0	175.0	488.0	433.0	383.0	358.0	30.0	40.0	1.651	101.3
150.0	175.0	498.0	438.0	383.0	358.0	35.0	45.0	1.676	102.3
160.0	175.0	508.0	443.0	383.0	358.0	40.0	50.0	1.701	103.3
170.0	175.0	518.0	448.0	383.0	358.0	45.0	55.0	1.726	104.2
180.0	175.0	528.0	453.0	383.0	358.0	50.0	60.0	1.751	105.1
190.0	175.0	538.0	458.0	383.0	358.0	55.0	65.0	1.761	106.1
200.0	175.0	548.0	463.0	383.0	358.0	60.0	70.0	1.771	107.0
210.0	175.0	558.0	468.0	383.0	358.0	65.0	75.0	1.773	107.9
220.0	175.0	568.0	473.0	383.0	358.0	70.0	80.0	1.773	108.8
230.0	175.0	578.0	478.0	383.0	358.0	75.0	85.0	1.773	109.7
240.0	175.0	588.0	483.0	383.0	358.0	80.0	90.0	1.773	110.6
250.0	175.0	598.0	488.0	383.0	358.0	85.0	95.0	1.773	111.5

Table A-1: page 1

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HI	OD	L1	W	S1	L2	W1	S2	L3	W2	S3	L4
(mm)	(mm)	(mm)	(mm)	(mm)	(mm)	(mm)	(mm)	(mm)	(mm)	(mm)	(mm)
100.0	200.0	448.0	413.0	358.0	10.0	20.0	1.473	1.473	80.0	74.4	69.6
110.0	200.0	458.0	418.0	358.0	15.0	25.0	1.499	1.499	80.8	74.5	68.9
120.0	200.0	468.0	423.0	358.0	20.0	30.0	1.499	1.499	81.6	74.5	68.2
130.0	200.0	478.0	428.0	358.0	25.0	35.0	1.524	1.524	82.4	74.6	67.6
140.0	200.0	488.0	433.0	358.0	30.0	40.0	1.524	1.524	83.2	74.7	67.0
150.0	200.0	498.0	438.0	358.0	35.0	45.0	1.549	1.549	84.0	74.8	66.4
160.0	200.0	508.0	443.0	358.0	40.0	50.0	1.575	1.575	84.8	74.9	65.8
170.0	200.0	518.0	448.0	358.0	45.0	55.0	1.575	1.575	85.5	75.0	65.2
180.0	200.0	528.0	453.0	358.0	50.0	60.0	1.600	1.600	86.3	75.1	64.6
190.0	200.0	538.0	458.0	358.0	55.0	65.0	1.600	1.600	87.1	75.2	64.1
200.0	200.0	548.0	463.0	358.0	60.0	70.0	1.626	1.626	87.8	75.3	63.5
210.0	200.0	558.0	468.0	358.0	65.0	75.0	1.626	1.626	88.5	75.4	63.0
220.0	200.0	568.0	473.0	358.0	70.0	80.0	1.651	1.651	89.3	75.5	62.5
230.0	200.0	578.0	478.0	358.0	75.0	85.0	1.651	1.651	90.0	75.7	62.0
240.0	200.0	588.0	483.0	358.0	80.0	90.0	1.676	1.676	90.7	75.8	61.5
250.0	200.0	598.0	488.0	358.0	85.0	95.0	1.676	1.676	91.5	75.9	61.1
100.0	225.0	448.0	413.0	358.0	10.0	20.0	1.397	1.397	67.4	62.7	58.7
110.0	225.0	458.0	418.0	358.0	15.0	25.0	1.422	1.422	68.1	62.7	58.1
120.0	225.0	468.0	423.0	358.0	20.0	30.0	1.422	1.422	68.7	62.8	57.6
130.0	225.0	478.0	428.0	358.0	25.0	35.0	1.448	1.448	69.4	62.9	57.0
140.0	225.0	488.0	433.0	358.0	30.0	40.0	1.448	1.448	70.0	62.9	56.5
150.0	225.0	498.0	438.0	358.0	35.0	45.0	1.473	1.473	70.7	63.0	56.0
160.0	225.0	508.0	443.0	358.0	40.0	50.0	1.473	1.473	71.3	63.1	55.5
170.0	225.0	518.0	448.0	358.0	45.0	55.0	1.499	1.499	72.0	63.2	55.0
180.0	225.0	528.0	453.0	358.0	50.0	60.0	1.499	1.499	72.6	63.3	54.5
190.0	225.0	538.0	458.0	358.0	55.0	65.0	1.524	1.524	73.2	63.3	54.1
200.0	225.0	548.0	463.0	358.0	60.0	70.0	1.524	1.524	73.8	63.4	53.6
210.0	225.0	558.0	468.0	358.0	65.0	75.0	1.549	1.549	74.4	63.5	53.2
220.0	225.0	568.0	473.0	358.0	70.0	80.0	1.549	1.549	75.1	63.6	52.8
230.0	225.0	578.0	478.0	358.0	75.0	85.0	1.575	1.575	75.7	63.7	52.4
240.0	225.0	588.0	483.0	358.0	80.0	90.0	1.575	1.575	76.3	63.8	52.0
250.0	225.0	598.0	488.0	358.0	85.0	95.0	1.600	1.600	76.9	63.9	51.6

Table A-1: page 2

Date: 8/13/92 Time: 8:22 Filename: Support analysis cases (sl)

Table A-1: Data 3

Date: 8/13/2022 Time: 8:22 Filename: Support analysis cases (all)												Net						
Ht	OD	L1			L2			L3			L4			Q1	Q2	Q3	Q4	
		[mm]	[mm]	[mm]	[mm]	[mm]	[mm]	[mm]	[mm]	[mm]	[mm]	[mm]	[mm]	[mm]	[mm]	[mm]	[mm]	
100.0	448.0	413.0	365.0	10.0	20.0	10.0	1,219	1,219	44.4	41.5	38.9	13.584	0.824	0.018	523.1	746.6	0.353	26.6
110.0	458.0	418.0	368.0	15.0	25.0	10.0	1,245	1,245	44.9	41.5	38.5	9.029	0.662	0.018	395.0	746.6	0.439	23.8
120.0	468.0	423.0	368.0	20.0	30.0	10.0	1,245	1,245	45.3	41.5	38.2	6.778	0.554	0.018	322.9	746.6	0.526	21.7
130.0	478.0	428.0	368.0	25.0	35.0	10.0	1,270	1,270	45.7	41.6	37.8	5.439	0.477	0.018	275.8	746.6	0.612	20.2
140.0	488.0	433.0	368.0	30.0	40.0	10.0	1,270	1,270	46.1	41.6	37.5	4.553	0.419	0.018	242.3	746.6	0.699	18.9
150.0	498.0	438.0	368.0	35.0	45.0	10.0	1,295	1,295	46.5	41.6	37.2	3.923	0.373	0.019	217.6	746.6	0.785	17.8
160.0	508.0	443.0	368.0	40.0	50.0	10.0	1,295	1,295	46.9	41.7	36.9	3.453	0.337	0.019	198.1	746.6	0.871	16.9
170.0	518.0	448.0	368.0	45.0	55.0	10.0	1,321	1,321	47.3	41.7	36.5	3.088	0.307	0.019	182.7	746.6	0.958	16.1
180.0	528.0	453.0	368.0	50.0	60.0	10.0	1,321	1,321	47.7	41.8	36.2	2.797	0.283	0.019	169.9	746.6	1.046	15.4
190.0	538.0	458.0	368.0	55.0	66.0	10.0	1,321	1,321	48.1	41.8	35.9	2.560	0.262	0.019	159.1	746.6	1.133	14.8
200.0	548.0	463.0	368.0	60.0	70.0	10.0	1,346	1,346	48.5	41.9	35.7	2.362	0.243	0.019	150.5	746.6	1.222	14.3
210.0	558.0	468.0	368.0	65.0	73.0	10.0	1,346	1,346	48.9	42.0	35.4	2.195	0.228	0.019	142.7	746.6	1.308	13.8
220.0	568.0	473.0	368.0	70.0	80.0	10.0	1,372	1,372	49.3	42.0	35.1	2.052	0.214	0.020	136.4	746.6	1.397	13.3
230.0	578.0	478.0	368.0	75.0	85.0	10.0	1,372	1,372	49.7	42.1	34.9	1.929	0.202	0.020	130.4	746.6	1.486	12.9
240.0	588.0	483.0	368.0	80.0	90.0	10.0	1,397	1,397	50.1	42.1	34.6	1.721	0.191	0.020	125.1	746.6	1.577	12.6
250.0	598.0	488.0	368.0	85.0	95.0	10.0	1,397	1,397	50.4	42.2	34.3	1.725	0.181	0.020	120.9	746.6	1.669	12.2

Table A-1: page 4

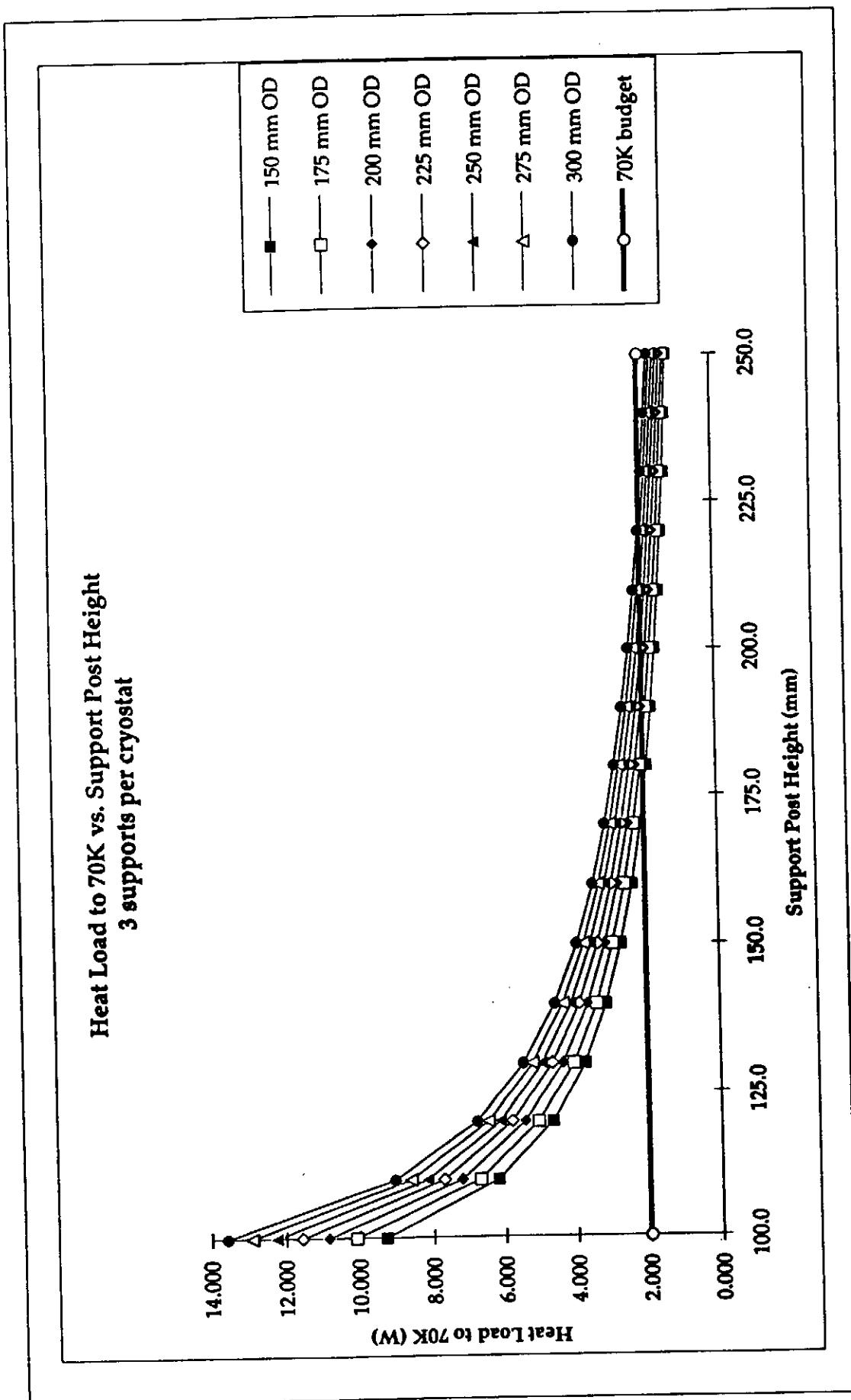


Figure A-1

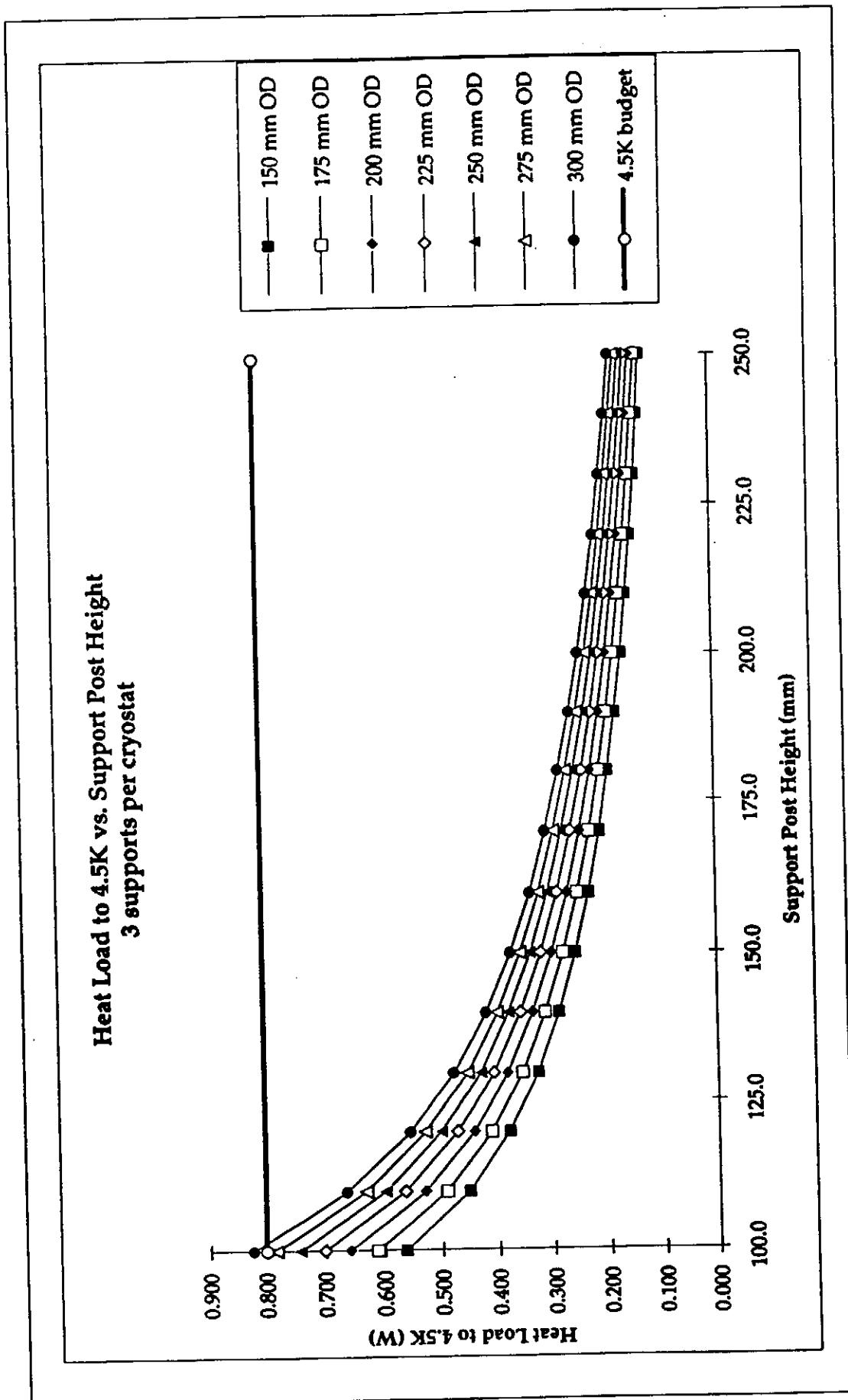


Figure A-2

Date: 8/13/92 Time: 8:09 Filename: Support analysis cases (si)

TESLA-Report 1994-01

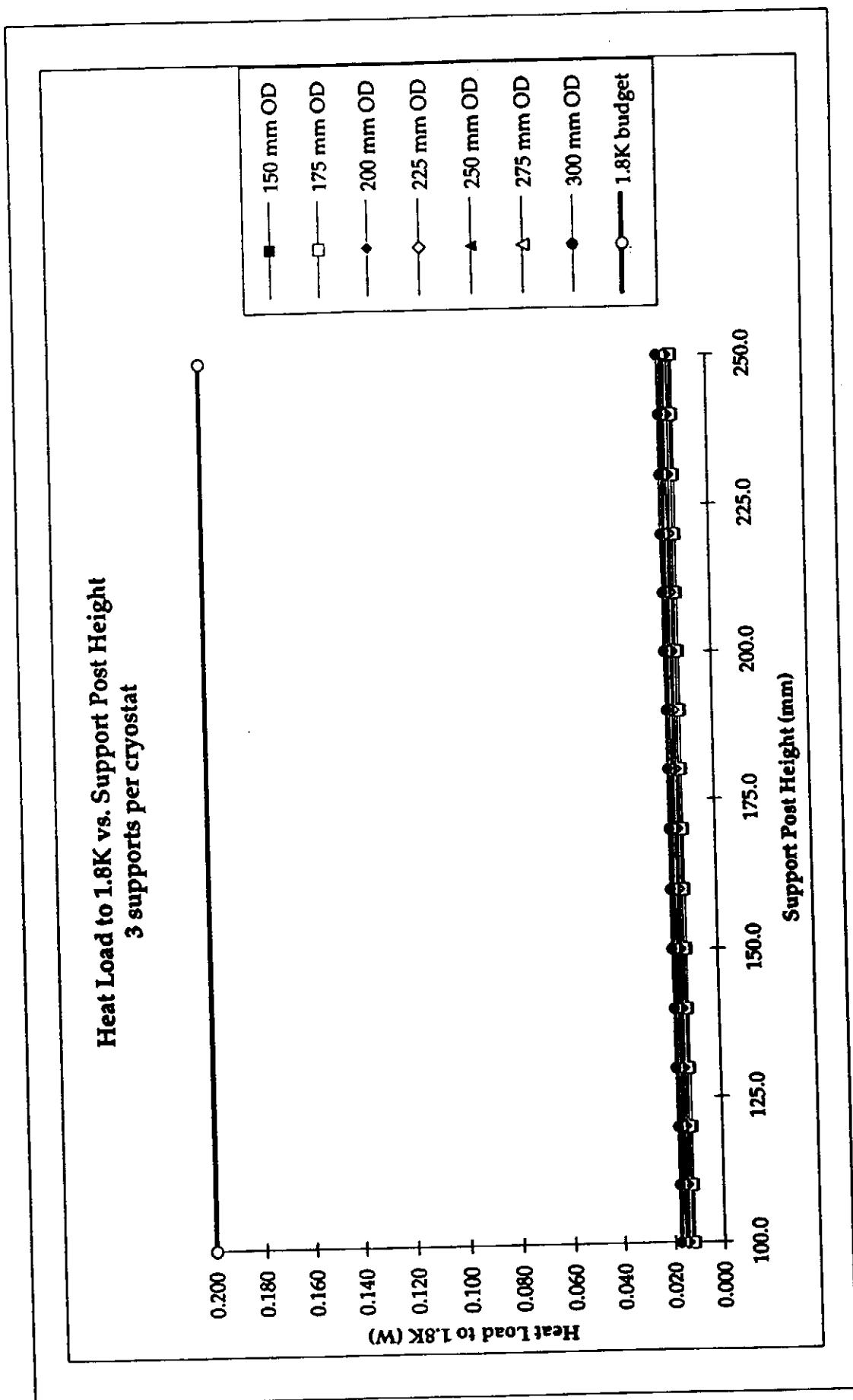


Figure A-3

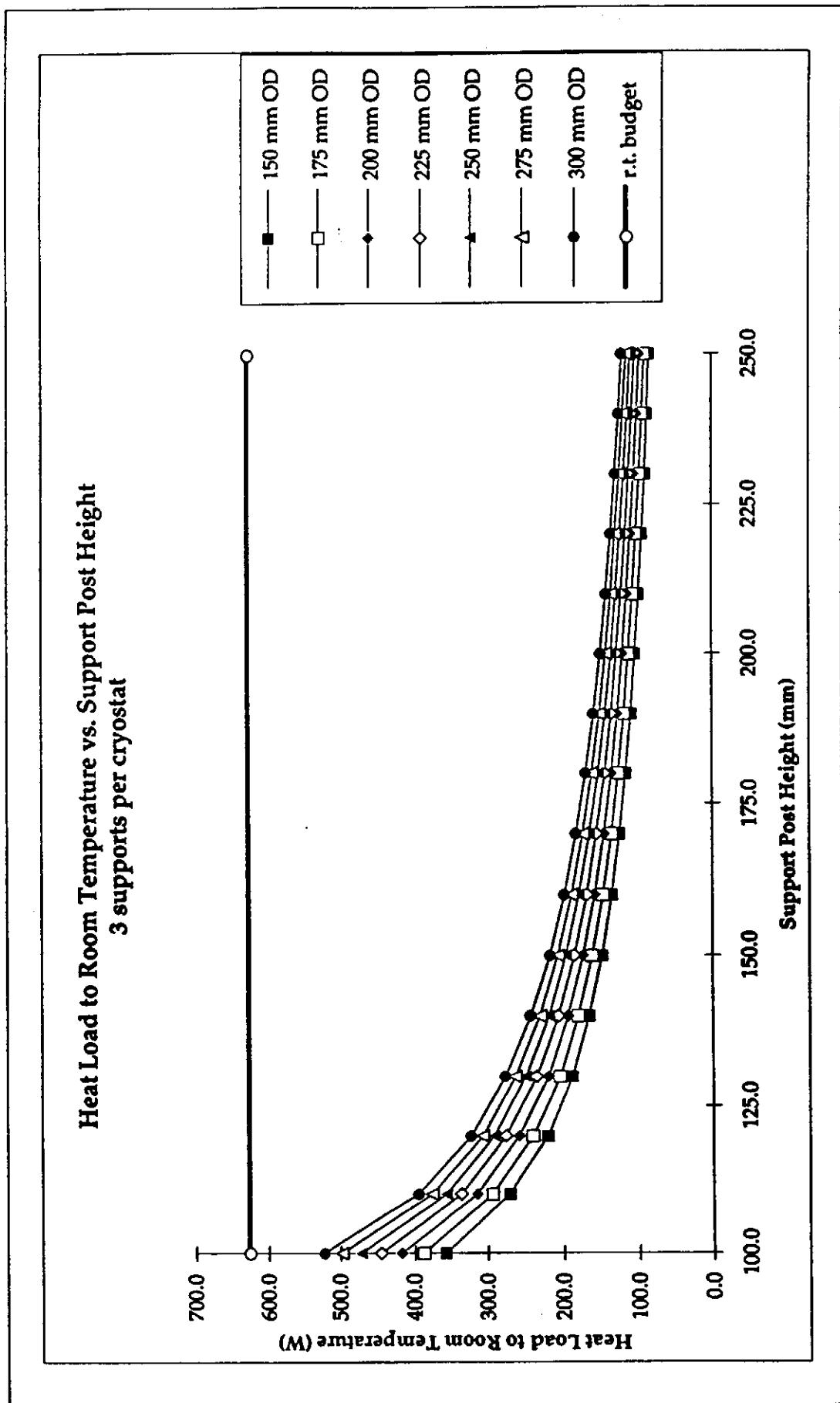


Figure A-4

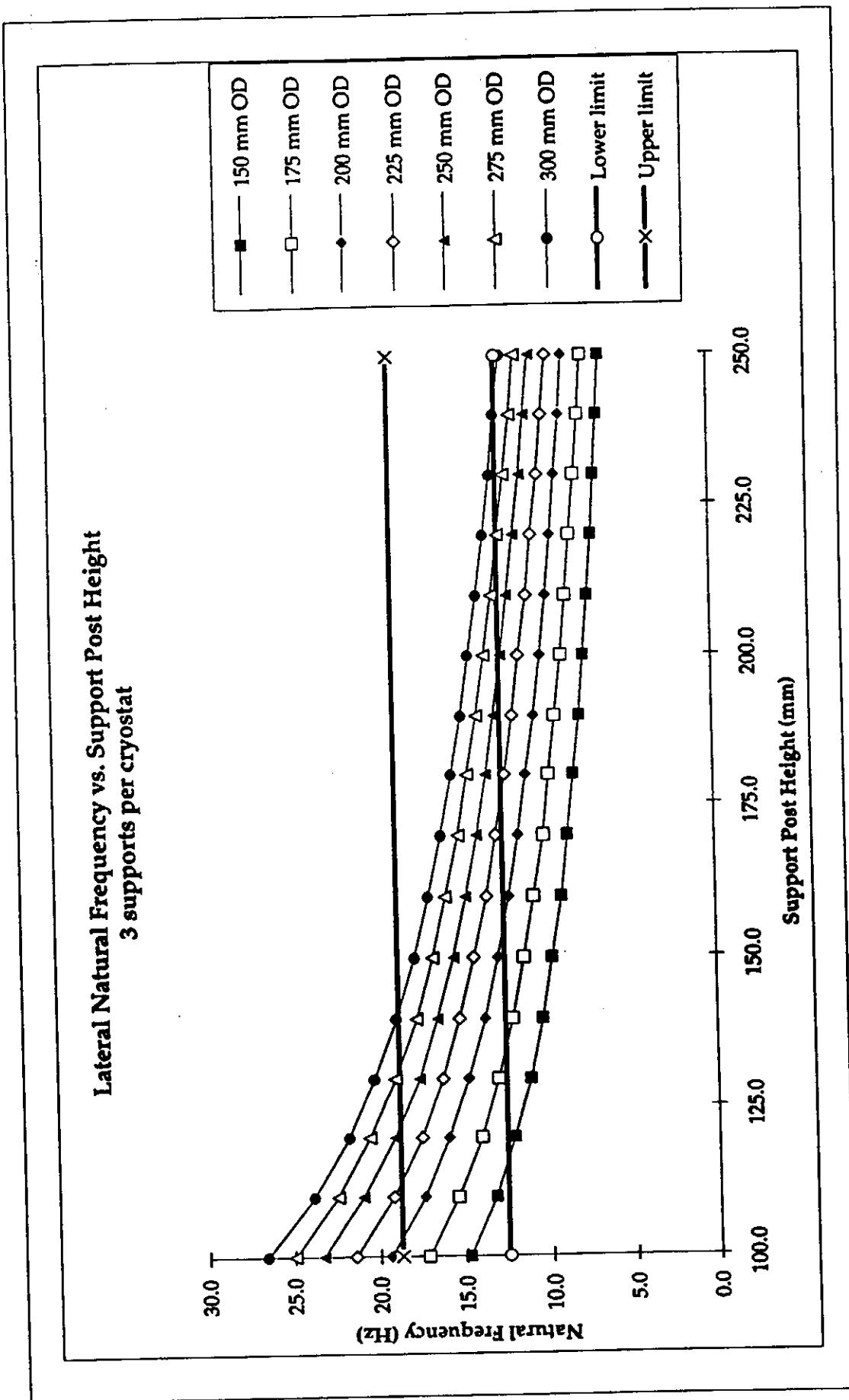


Figure A-5

## **APPENDIX B**

### **Thermal and Structural Analysis Results 4 Supports per Cryostat**

TESLA Faraday Support Case  
Report per Total  
T. Nees, Fermilab - August 1992

Refrigeration System WELL TO WELL			
T °K	Q W/m	T °K	Q W/m
70	16	70	1.50
4.5	20%	4.5	0.60
1.8	10%	1.8	0.15
			249

Heat Load Budget (W)			
T °K	Q W/m	T °K	Q W/m
70	16	70	1.50
4.5	20%	4.5	0.60
1.8	10%	1.8	0.15
			249

Data for Optimizing Pad Effectiveness											
Ht (mm)	Wd (mm)	L1 (mm)	L2 (mm)	L3 (mm)	L4 (mm)	W1 (mm)	W2 (mm)	Q1a (W)	Q1b (W)	Q2a (W)	Q2b (W)
100.0	448.0	413.0	358.0	10.0	20.0	10.0	1.473	1.473	105.7	98.1	91.6
110.0	458.0	418.0	358.0	15.0	25.0	10.0	1.473	1.473	106.8	98.2	90.6
120.0	468.0	423.0	358.0	20.0	30.0	10.0	1.499	1.499	107.9	98.3	89.8
130.0	478.0	428.0	358.0	25.0	35.0	10.0	1.499	1.499	109.0	98.4	88.9
140.0	488.0	433.0	358.0	30.0	40.0	10.0	1.524	1.524	110.1	98.5	88.1
150.0	498.0	438.0	358.0	35.0	45.0	10.0	1.549	1.549	111.1	98.7	87.2
160.0	508.0	443.0	358.0	40.0	50.0	10.0	1.549	1.549	112.2	98.8	86.4
170.0	518.0	448.0	358.0	45.0	55.0	10.0	1.575	1.575	113.2	98.9	85.7
180.0	528.0	453.0	358.0	50.0	60.0	10.0	1.575	1.575	114.3	99.1	84.9
190.0	538.0	458.0	358.0	55.0	65.0	10.0	1.600	1.600	115.3	99.2	84.2
200.0	548.0	463.0	358.0	60.0	70.0	10.0	1.600	1.600	116.3	99.4	83.5
210.0	558.0	468.0	358.0	65.0	75.0	10.0	1.626	1.626	117.3	99.6	82.8
220.0	568.0	473.0	358.0	70.0	80.0	10.0	1.626	1.626	118.3	99.7	82.1
230.0	578.0	478.0	358.0	75.0	85.0	10.0	1.651	1.651	119.3	99.9	81.5
240.0	588.0	483.0	358.0	80.0	90.0	10.0	1.676	1.676	120.3	100.1	80.8
250.0	598.0	488.0	358.0	85.0	95.0	10.0	1.676	1.676	121.3	100.2	80.2
100.0	448.0	413.0	358.0	10.0	20.0	10.0	1.372	1.372	84.2	78.2	73.1
110.0	458.0	418.0	358.0	15.0	25.0	10.0	1.372	1.372	85.1	78.3	72.3
120.0	468.0	423.0	358.0	20.0	30.0	10.0	1.397	1.397	85.9	78.4	71.6
130.0	478.0	428.0	358.0	25.0	35.0	10.0	1.397	1.397	86.8	78.4	70.9
140.0	488.0	433.0	358.0	30.0	40.0	10.0	1.422	1.422	87.6	78.5	70.3
150.0	498.0	438.0	358.0	35.0	45.0	10.0	1.422	1.422	88.4	78.6	69.6
160.0	508.0	443.0	358.0	40.0	50.0	10.0	1.448	1.448	89.3	78.7	69.0
170.0	518.0	448.0	358.0	45.0	55.0	10.0	1.448	1.448	90.1	78.8	68.4
180.0	528.0	453.0	358.0	50.0	60.0	10.0	1.473	1.473	90.9	78.9	67.8
190.0	538.0	458.0	358.0	55.0	65.0	10.0	1.473	1.473	91.7	79.1	67.2
200.0	548.0	463.0	358.0	60.0	70.0	10.0	1.499	1.499	92.5	79.2	66.7
210.0	558.0	468.0	358.0	65.0	75.0	10.0	1.499	1.499	93.3	79.3	66.1
220.0	568.0	473.0	358.0	70.0	80.0	10.0	1.524	1.524	94.1	79.4	65.6
230.0	578.0	478.0	358.0	75.0	85.0	10.0	1.524	1.524	94.8	79.6	65.1
240.0	588.0	483.0	358.0	80.0	90.0	10.0	1.549	1.549	95.6	79.7	64.5
250.0	598.0	488.0	358.0	85.0	95.0	10.0	1.549	1.549	96.4	79.8	64.0

Table B-1: page 1

Filename: Support analysis cases (a)													
H1 (mm)	OD (mm)	T1 (mm)	L1 (mm)	L2 (mm)	L3 (mm)	L4 (mm)	L5 (mm)	L6 (mm)	L7 (mm)	L8 (mm)	L9 (mm)	L10 (mm)	Q1 (N)
100.0	448.0	413.0	383.0	358.0	10.0	20.0	10.0	1.270	69.2	64.3	60.2	9.383	0.569
100.0	200.0	448.0	418.0	383.0	15.0	25.0	10.0	1.295	69.9	64.4	59.6	6.240	0.458
110.0	200.0	458.0	418.0	383.0	15.0	20.0	10.0	1.295	1.295	70.6	64.5	59.0	4.686
120.0	200.0	468.0	423.0	383.0	20.0	30.0	10.0	1.295	1.295	71.3	64.5	58.4	3.762
130.0	200.0	478.0	428.0	383.0	25.0	35.0	10.0	1.321	1.321	72.0	64.6	57.9	3.151
140.0	200.0	488.0	433.0	383.0	30.0	40.0	10.0	1.321	1.321	72.6	64.7	57.4	2.716
150.0	200.0	498.0	438.0	383.0	35.0	45.0	10.0	1.346	1.346	73.3	64.8	56.9	2.391
160.0	200.0	508.0	443.0	383.0	40.0	50.0	10.0	1.346	1.346	74.0	64.8	56.4	2.140
170.0	200.0	518.0	448.0	383.0	45.0	55.0	10.0	1.372	1.372	74.6	64.9	55.9	1.939
180.0	200.0	528.0	453.0	383.0	50.0	60.0	10.0	1.372	1.372	75.3	65.0	55.4	1.775
190.0	200.0	538.0	458.0	383.0	55.0	65.0	10.0	1.397	1.397	75.9	65.1	54.9	1.638
200.0	200.0	548.0	463.0	383.0	60.0	70.0	10.0	1.397	1.397	76.6	65.2	54.5	1.523
210.0	200.0	558.0	468.0	383.0	65.0	75.0	10.0	1.422	1.422	77.2	65.3	54.1	1.424
220.0	200.0	568.0	473.0	383.0	70.0	80.0	10.0	1.422	1.422	77.8	65.4	53.6	1.339
230.0	200.0	578.0	478.0	383.0	75.0	85.0	10.0	1.448	1.448	78.4	65.5	53.2	1.264
240.0	200.0	588.0	483.0	383.0	80.0	90.0	10.0	1.448	1.448	79.1	65.6	52.8	1.198
250.0	200.0	598.0	488.0	383.0	85.0	95.0	10.0	1.448	1.448	79.7	65.6	52.4	1.134
100.0	225.0	448.0	413.0	383.0	358.0	10.0	20.0	1.219	1.219	58.3	54.2	50.8	10.010
110.0	225.0	458.0	418.0	383.0	358.0	15.0	25.0	1.219	1.219	58.9	54.3	50.3	6.656
120.0	225.0	468.0	423.0	383.0	358.0	20.0	30.0	1.245	1.245	59.4	54.3	49.8	4.998
130.0	225.0	478.0	428.0	383.0	358.0	25.0	35.0	1.245	1.245	60.0	54.4	49.3	4.012
140.0	225.0	488.0	433.0	383.0	358.0	30.0	40.0	1.270	1.270	60.6	54.4	48.9	3.359
150.0	225.0	498.0	438.0	383.0	358.0	35.0	45.0	1.270	1.270	61.1	54.5	48.4	2.896
160.0	225.0	508.0	443.0	383.0	358.0	40.0	50.0	1.270	1.270	61.7	54.6	48.0	2.549
170.0	225.0	518.0	448.0	383.0	358.0	45.0	55.0	1.295	1.295	62.2	54.6	47.6	2.281
180.0	225.0	528.0	453.0	383.0	358.0	50.0	60.0	1.295	1.295	62.8	54.7	47.2	2.066
190.0	225.0	538.0	458.0	383.0	358.0	55.0	65.0	1.321	1.321	63.3	54.8	46.8	1.892
200.0	225.0	548.0	463.0	383.0	358.0	60.0	70.0	1.321	1.321	63.9	54.9	46.4	1.746
210.0	225.0	558.0	468.0	383.0	358.0	65.0	75.0	1.346	1.346	64.4	54.9	46.0	1.623
220.0	225.0	568.0	473.0	383.0	358.0	70.0	80.0	1.346	1.346	64.9	55.0	45.7	1.518
230.0	225.0	578.0	478.0	383.0	358.0	75.0	85.0	1.372	1.372	65.4	55.1	45.3	1.426
240.0	225.0	588.0	483.0	383.0	358.0	80.0	90.0	1.372	1.372	66.0	55.2	44.9	1.347
250.0	225.0	598.0	488.0	383.0	358.0	85.0	95.0	1.372	1.372	66.5	55.3	44.6	1.276

Table B-1: page 2

Date: 8/13/92 Time: 8:25 Filename: Support analysis cases (s)

Hr	OD 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Date: 8/13/92 Time: 8:25 Filename: Support analysis cases (all)

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Date: 8/13/92 Time: 8:15 Filename: Support analysis cases (sl)

TESLA-Report 1994-01

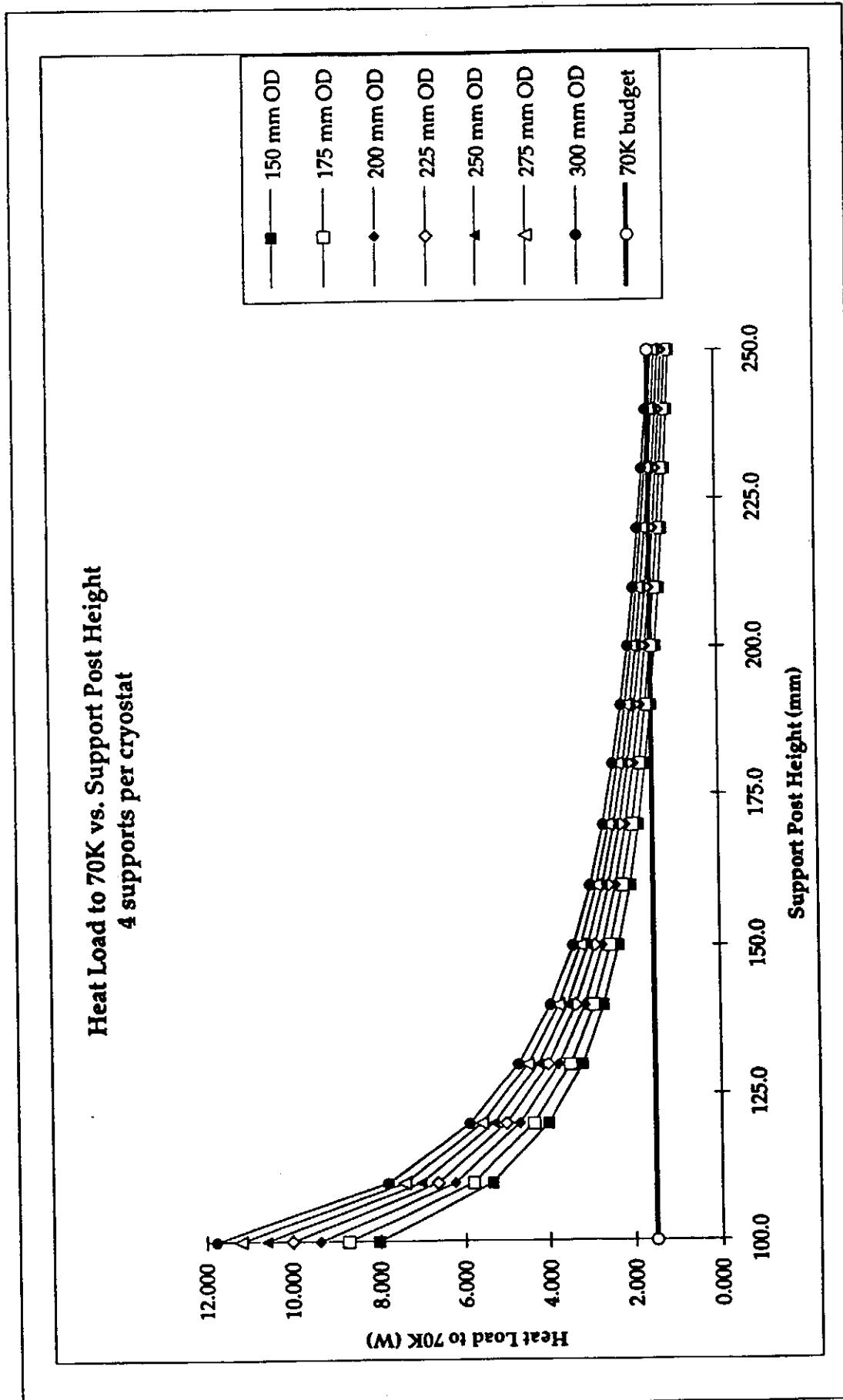


Figure B-1

Date: 8/13/92 Time: 8:15 Filename: Support analysis cases (si)

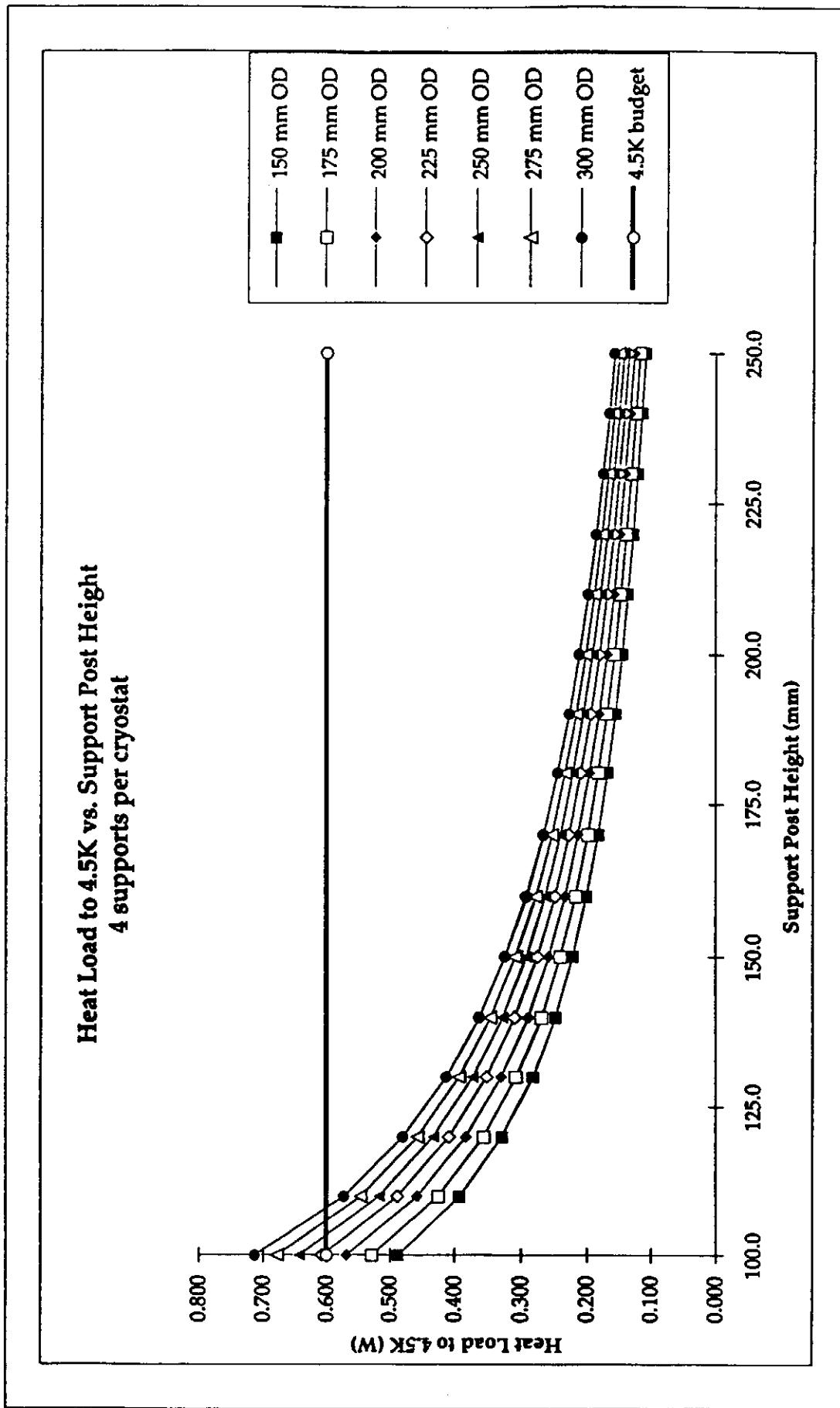
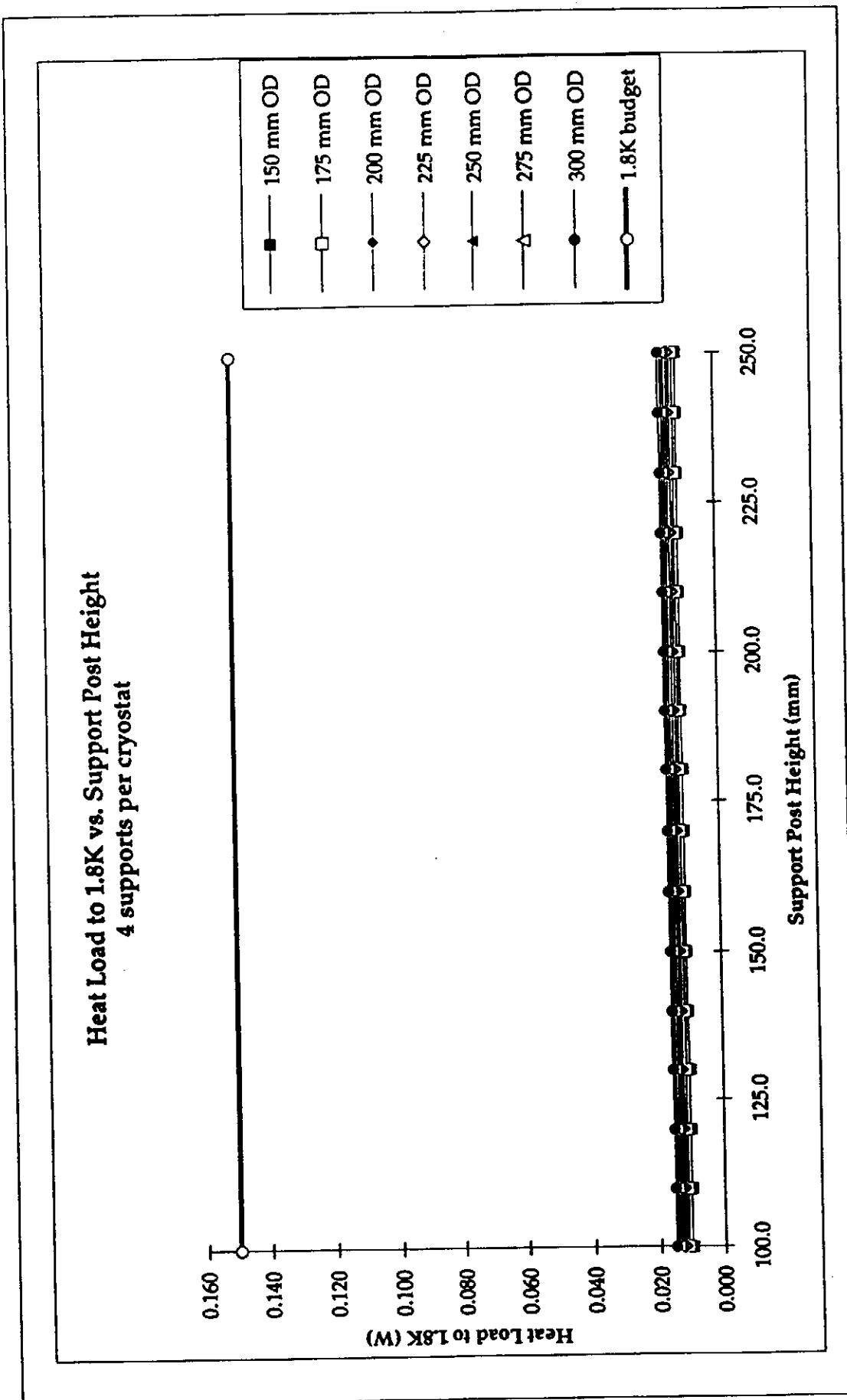


Figure B-2

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**Figure B-3**

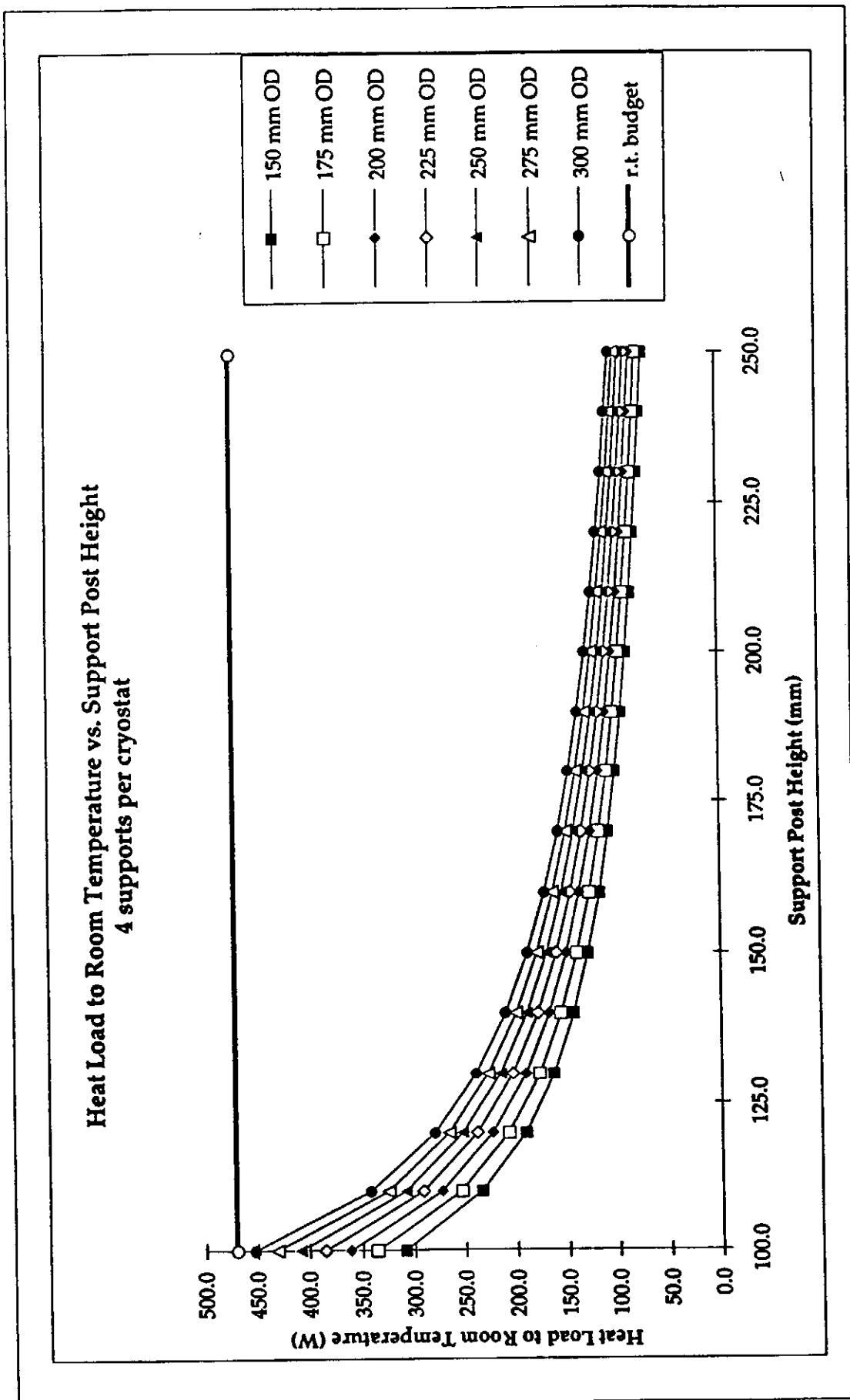


Figure B-4

Date: 8/13/92 Time: 8:15 Filename: Support analysis cases (sl)

TESLA-Report 1994-01

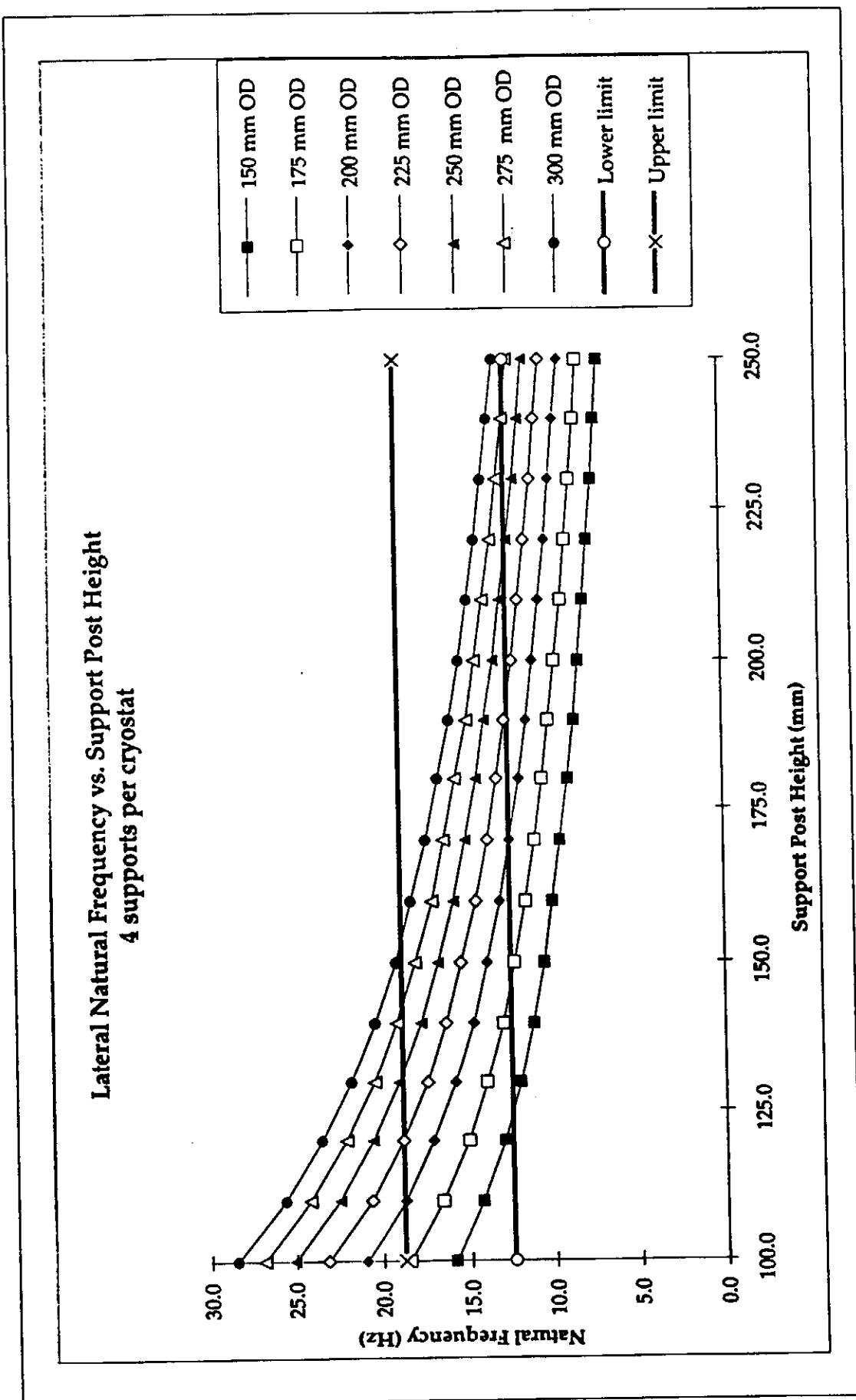


Figure B-5